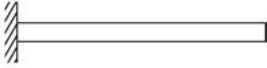
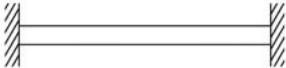
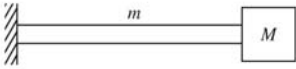



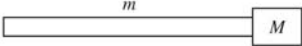
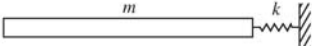


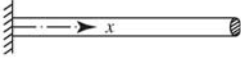
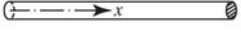
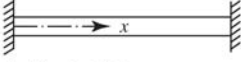
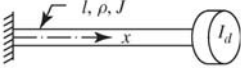

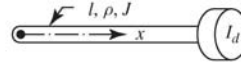
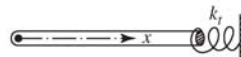
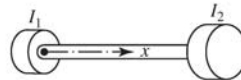
۱. جدول فرکانسهای طبیعی محوری میله‌ها

Table 9.1 Boundary Conditions of a Bar in Longitudinal Vibration

End conditions of the bar	Boundary conditions	Frequency equation	Mode shapes (normal functions)	Natural frequencies of vibration
1. Fixed-free 	$u(0, t) = 0$ $\frac{\partial u}{\partial x}(l, t) = 0$	$\cos \frac{\omega l}{c} = 0$	$U_n(x) = C_n \sin \frac{(2n+1)\pi x}{2l}$	$\omega_n = \frac{(2n+1)\pi c}{2l}$, $n = 0, 1, 2, \dots$
2. Fixed-fixed 	$u(0, t) = 0$ $u(l, t) = 0$	$\sin \frac{\omega l}{c} = 0$	$U_n(x) = C_n \sin \frac{n\pi x}{l}$	$\omega_n = \frac{n\pi c}{l}$, $n = 1, 2, 3, \dots$
3. Fixed-attached mass 	$u(0, t) = 0$ $EA \frac{\partial u}{\partial x}(l, t) = -M \frac{\partial^2 u}{\partial t^2}(l, t)$	$\alpha \tan \alpha = \beta$ $\alpha = \frac{\omega l}{c}$ $\beta = \frac{m}{M}$	$U_n(x) = C_n \sin \frac{\omega_n x}{c}$	$\omega_n = \frac{\alpha_n c}{l}$, $n = 1, 2, 3, \dots$
4. Fixed-attached spring 	$u(0, t) = 0$ $EA \frac{\partial u}{\partial x}(l, t) = -ku(l, t)$	$\alpha \tan \alpha = -\gamma$ $\alpha = \frac{\omega l}{c}$ $\gamma = \frac{m\omega^2}{k}$	$U_n(x) = C_n \sin \frac{\omega_n x}{c}$	$\omega_n = \frac{\alpha_n c}{l}$, $n = 1, 2, 3, \dots$
5. Fixed-attached spring and mass 	$u(0, t) = 0$ $EA \frac{\partial u}{\partial x}(l, t) = -M \frac{\partial^2 u}{\partial t^2}(l, t) - ku(l, t)$	$\alpha \cot \alpha = \frac{\alpha^2}{\beta} - \frac{k}{k_0}$ $\beta = \frac{m}{M}$ $k_0 = \frac{AE}{l}$	$U_n(x) = C_n \sin \frac{\omega_n x}{c}$	$\omega_n = \frac{\alpha_n c}{l}$, $n = 1, 2, 3, \dots$
6. Free-free 	$\frac{\partial u}{\partial x}(0, t) = 0$ $\frac{\partial u}{\partial x}(l, t) = 0$	$\sin \frac{\omega l}{c} = 0$	$U_n(x) = C_n \cos \frac{n\pi x}{l}$	$\omega_n = \frac{n\pi c}{l}$, $n = 0, 1, 2, \dots$
7. Free-attached mass 	$\frac{\partial u}{\partial x}(0, t) = 0$ $EA \frac{\partial u}{\partial x}(l, t) = -M \frac{\partial^2 u}{\partial t^2}(l, t)$	$\tan \alpha = -\alpha\beta$	$U_n(x) = C_n \cos \frac{\omega_n x}{c}$	$\omega_n = \frac{\alpha_n c}{l}$, $n = 1, 2, 3, \dots$
8. Free-attached spring 	$\frac{\partial u}{\partial x}(0, t) = 0$ $EA \frac{\partial u}{\partial x}(l, t) = -ku(l, t)$	$\alpha \cot \alpha = \delta$ $\delta = \frac{AE}{lk}$	$U_n(x) = C_n \cos \frac{\omega_n x}{c}$	$\omega_n = \frac{\alpha_n c}{l}$, $n = 1, 2, 3, \dots$

۲. جدول فرکانسهای طبیعی پیچشی میله‌ها

Table 10.1 Boundary Conditions of a Uniform Shaft in Torsional Vibration

End conditions of shaft	Boundary conditions	Frequency equation	Mode shape (normal function)	Natural frequencies
1. Fixed-free 	$\theta(0, t) = 0$ $\frac{\partial \theta}{\partial x}(l, t) = 0$	$\cos \frac{\omega l}{c} = 0$	$\Theta_n(x) = C_n \sin \frac{(2n+1)\pi x}{2l}$	$\omega_n = \frac{(2n+1)\pi c}{2l}$, $n = 0, 1, 2, \dots$
2. Free-free 	$\frac{\partial \theta}{\partial x}(0, t) = 0$ $\frac{\partial \theta}{\partial x}(l, t) = 0$	$\sin \frac{\omega l}{c} = 0$	$\Theta_n(x) = C_n \cos \frac{n\pi x}{l}$	$\omega_n = \frac{n\pi c}{l}$, $n = 0, 1, 2, \dots$
3. Fixed-fixed 	$\theta(0, t) = 0$ $\theta(l, t) = 0$	$\sin \frac{\omega l}{c} = 0$	$\Theta_n(x) = C_n \sin \frac{n\pi x}{l}$	$\omega_n = \frac{n\pi c}{l}$, $n = 1, 2, 3, \dots$
4. Fixed-disk 	$\theta(0, t) = 0$ $GJ \frac{\partial \theta}{\partial x}(l, t) = -I_d \frac{\partial^2 \theta}{\partial t^2}(l, t)$	$\alpha \tan \alpha = \beta$ $\alpha = \frac{\omega l}{c}$ $\beta = \frac{\rho J l}{I_d}$	$\Theta_n(x) = C_n \sin \frac{\omega_n x}{c}$	$\omega_n = \frac{\alpha_n c}{l}$, $n = 1, 2, 3, \dots$
5. Fixed-torsional spring 	$\theta(0, t) = 0$ $GJ \frac{\partial \theta}{\partial x}(l, t) = -k_t \theta(l, t)$	$\alpha \tan \alpha = -\beta$ $\alpha = \frac{\omega l}{c}$ $\beta = \frac{\omega^2 \rho J l}{k_t}$	$\Theta_n(x) = C_n \sin \frac{\omega_n x}{c}$	$\omega_n = \frac{\alpha_n c}{l}$, $n = 1, 2, 3, \dots$
6. Free-disk 	$\frac{\partial \theta}{\partial x}(0, t) = 0$ $GJ \frac{\partial \theta}{\partial x}(l, t) = -I_d \frac{\partial^2 \theta}{\partial t^2}(l, t)$	$\alpha \cot \alpha = -\beta$ $\alpha = \frac{\omega l}{c}$ $\beta = \frac{\rho J l}{I_d}$	$\Theta_n(x) = C_n \cos \frac{\omega_n x}{c}$	$\omega_n = \frac{\alpha_n c}{l}$, $n = 1, 2, 3, \dots$
7. Free-torsional spring 	$\frac{\partial \theta}{\partial x}(0, t) = 0$ $GJ \frac{\partial \theta}{\partial x}(l, t) = -k_t \theta(l, t)$	$\alpha \cot \alpha = \beta$ $\alpha = \frac{\omega l}{c}$ $\beta = \frac{\omega^2 \rho J l}{k_t}$	$\Theta_n(x) = C_n \cos \frac{\omega_n x}{c}$	$\omega_n = \frac{\alpha_n c}{l}$, $n = 1, 2, 3, \dots$
8. Disk-disk 	$GJ \frac{\partial \theta}{\partial x}(0, t) = I_1 \frac{\partial^2 \theta}{\partial t^2}(0, t)$ $GJ \frac{\partial \theta}{\partial x}(l, t) = -I_2 \frac{\partial^2 \theta}{\partial t^2}(l, t)$	$\tan \alpha = \frac{\alpha(\beta_1 + \beta_2)}{(\alpha^2 - \beta_1 \beta_2)}$ $\alpha = \frac{\omega l}{c}$ $\beta_1 = \frac{I_0}{I_1} = \frac{\rho J l}{I_1}$ $\beta_2 = \frac{I_0}{I_2} = \frac{\rho J l}{I_2}$	$\Theta_n(x) = C_n \left(\cos \frac{\alpha_n x}{l} - \frac{\alpha_n}{\beta_1} \sin \frac{\alpha_n x}{l} \right)$	$\omega_n = \frac{\alpha_n c}{l}$, $n = 1, 2, 3, \dots$

۳. مرجع

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